

①

$$F_x = 0$$

$$F_y = -\rho U \Gamma$$



no drag?

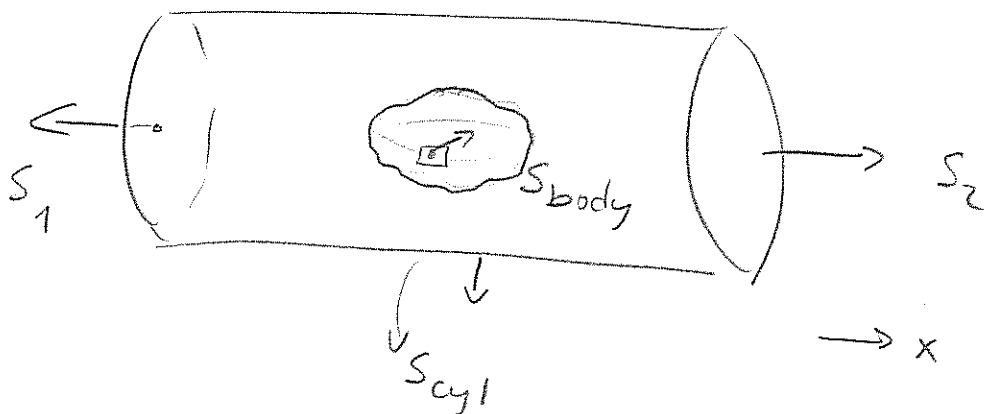
("D'Alembert paradox" — or theorem)

Also in 3d... momentum equation

(steady flow)

$$(\vec{u} \cdot \nabla) \vec{u} = -\nabla p$$

$$(1) \quad (u_i \partial_i) u_j = -\partial_j p, \quad \text{in index notation}$$



$$\int (1) \text{ over } V: \quad \underbrace{\rho \int dV (u_i \partial_i u_j)}_{\text{L.h.s.}} = - \underbrace{\int \partial_j p dV}_{\text{R.h.s.}}$$

$$\begin{aligned} \underline{\text{L.h.s.}} &= \rho \int dV u_i \partial_i u_j = \rho \int dV \partial_i (u_i u_j) = (\text{since } \partial_i u_i = 0) \\ &= \rho \oint_S d^2s^i u^i u^j = \rho \int_{S_{\text{cyl}}} d^2s^i u^i u^j + \\ &+ \rho \int_{S_1} d^2s^i u^i u^j + \rho \int_{S_2} d^2s^i u^i u^j + \rho \int_{S_{\text{cyl}}} d^2s^i u^i u^j \end{aligned}$$

(2)

on both $S_{cyl} \neq S_{body}$ \vec{u} is $\perp d^2\vec{S}$

so $d^2S^i u^i = 0$ on $S_{cyl} \neq S_{body}$

So we have

$$\underline{\underline{Lhs}} = \rho \int_{S_1} d^2S^i u^i u^j + \rho \int_{S_2} d^2S^i u^i u^j$$

$$\underline{\underline{r.h.s.}} = - \int_V \partial_j p dV = - \int_V \partial_i (\delta_{ij} p) dV =$$

$$= - \oint_S d^2S^i \delta^{ij} p =$$

$$= - \int_{S_{cyl}} d^2S^i \delta^{ij} p - \int_{S_{body}} d^2S^i \delta^{ij} p$$

$$- \int_{S_1} d^2S^i \delta^{ij} p - \int_{S_2} d^2S^i \delta^{ij} p$$

Take

(r.h.s.)^{j=x} component : along channel

$$- \int_V \partial_x p dV = - \int_{S_{cyl}} d^2S^x p - \int_{S_{body}} d^2S^x p - \int_{S_1} d^2S^x p - \int_{S_2} d^2S^x p$$

$\left(\begin{array}{l} \vec{d^2S}_{cyl} \text{ is } \perp \vec{x} \\ \text{so this} = 0 \end{array} \right)$ \downarrow (force)^x on body = (drag)^x

So we have

$$\begin{aligned}
 (\text{r.h.s.})^{j=x} &= -(\text{Drag})^x + \int_{S_1} d^2S p - \int_{S_2} d^2S p \\
 (\text{l.h.s.})^{j=x} &= \rho \int_{S_1} \underbrace{d^2S^x}_{-d^2S} u^x u^x + \rho \int_{S_2} \underbrace{d^2S^x}_{+d^2S} u^x u^x \\
 &= -\rho \int_{S_1} d^2S (u_x)^2 + \rho \int_{S_2} d^2S (u_x)^2
 \end{aligned}$$

equated the two:

$$\begin{aligned}
 (\text{Drag})^x &= - \int_{S_1} d^2S (\rho (u_x)^2 + p) \\
 &\quad + \int_{S_2} d^2S (\rho (u_x)^2 + p)
 \end{aligned}$$

take $S_2 \rightarrow +\infty$
 $S_2 \rightarrow -\infty$ } @ ∞ 's flow is unidirectional
 & uniform, pressure = p_0

By Bernoulli along streamlines

hence $\rho (u_x)^2 + p \Big|_{+\infty} = \rho (u_x)^2 + p \Big|_{-\infty}$

& there's no drag.

⇒ a steady uniform flow of ideal fluid

⇒ no drag

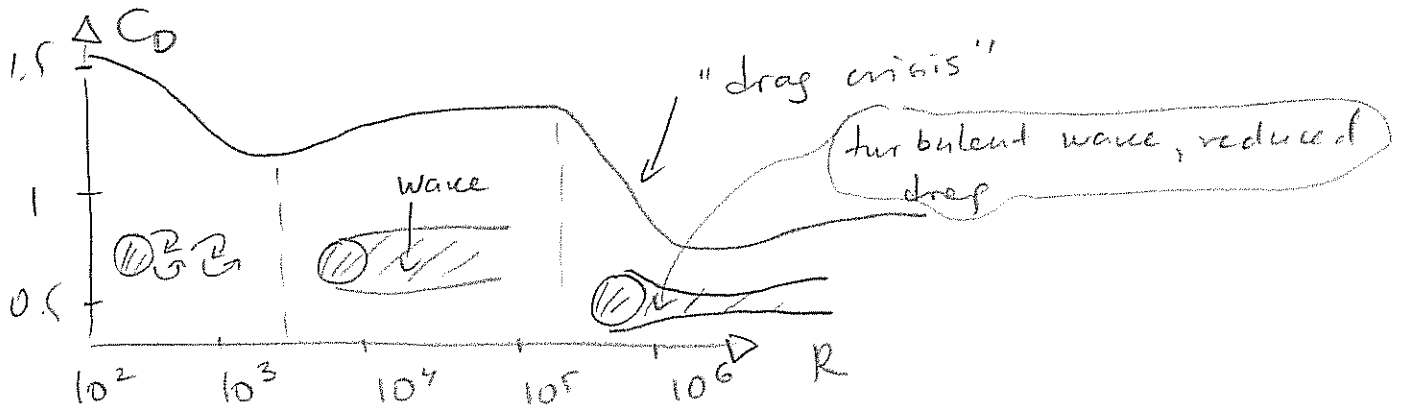
(D'Alembert)

different from real flow in that

there is no wake

recall $C_D = \frac{Drag}{\rho U^2 a}$ vs. $R = \frac{2aU}{\nu}$ $a = \text{radius}$

@ $R \rightarrow \infty$ expected $C_D \rightarrow 1$... but in reality



($C_D \not\rightarrow 1$ as $R \rightarrow \infty$ -- always \exists wake)

boundary layer & its separation