

Since $\vec{F} = -6\pi\mu a \vec{U}$

Use to measure μ

$m\ddot{z} = -mg + 6\pi\mu a \dot{z}$



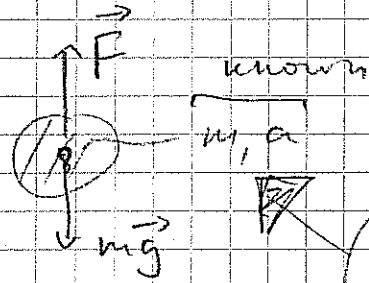
limiting velocity is

rhs = 0

$v_{lim} = \dot{z} = \frac{mg}{6\pi\mu a} \Rightarrow$

$\mu = \frac{mg}{6\pi a v_{lim}}$

(very good @ $R \ll 1$)
(up to $R \sim 0.3$ or so)



neglected buoyancy, trivial to include in eqn.

Can also solve rigid ellipsoid (DIY) :-)

For a body of arbitrary shape $F_i = 4\pi\mu P_{ij} U_j$

& flow field at large distances is like that for a sphere, i.e. axisymmetric w.r.t. P_{ij}, U_j -axis

tensor depending on shape of body

No general useful recipe for P_{ij} , however...

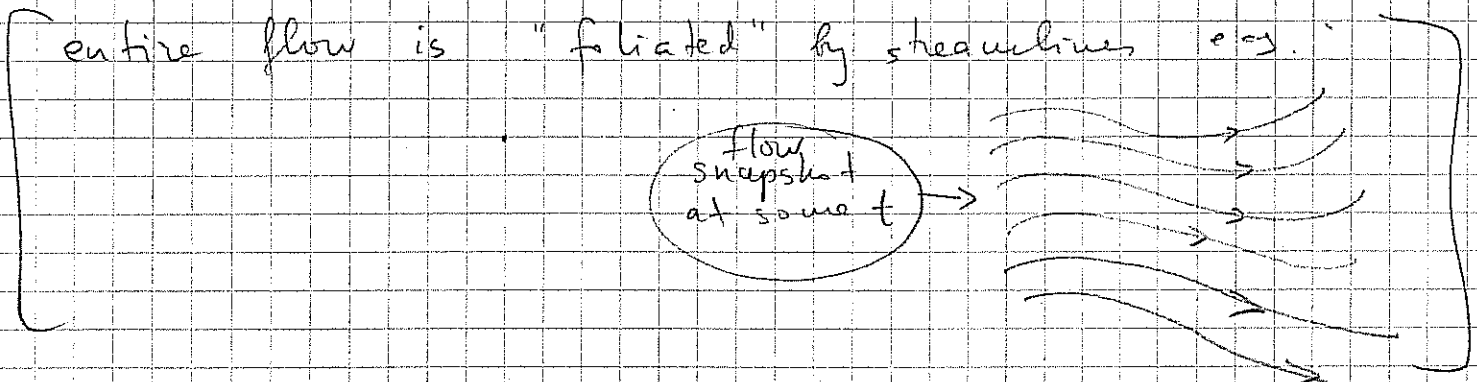
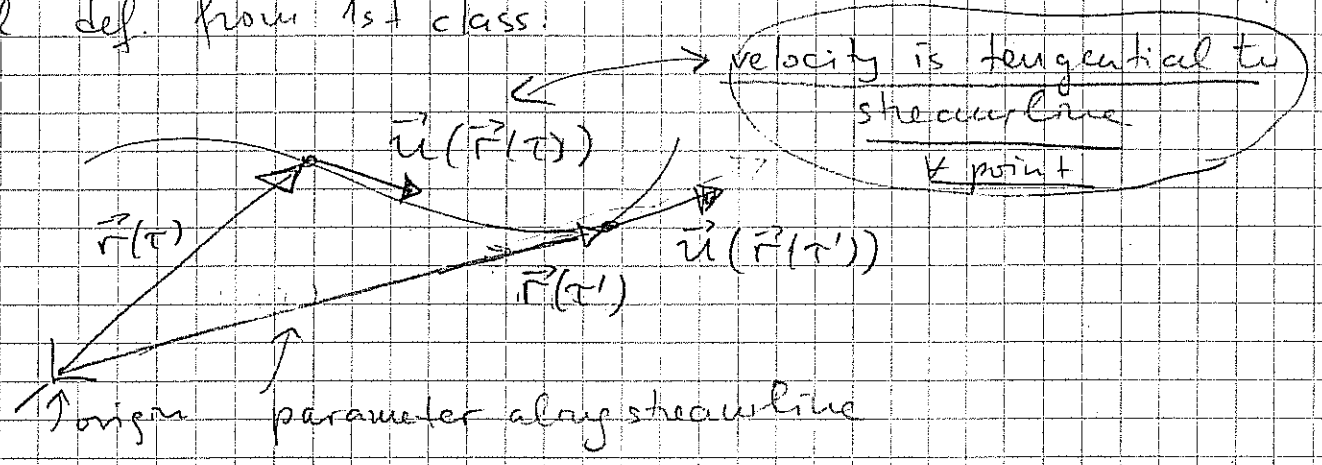
-> measure direction of force - find P_{ij}, U_j direction -

-> far away flow field is ~ "Stokeslet at origin"

picture on p 132 → "not so good" (interpret??)

* let's find streamlines of Stokes' flow

recall def. from 1st class.



i.e. "E.O.M" for streamlines:

$$\frac{d\vec{r}(\tau)}{d\tau} = \vec{u}(\vec{r}(\tau), t)$$

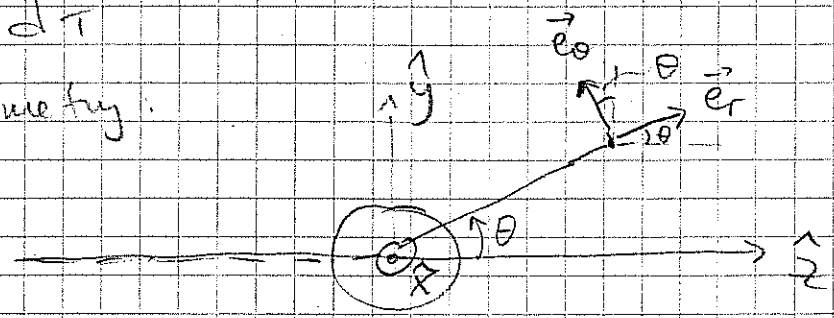
↑ fluid flow field

if flow NOT stationary, streamlines "wobble" - different t

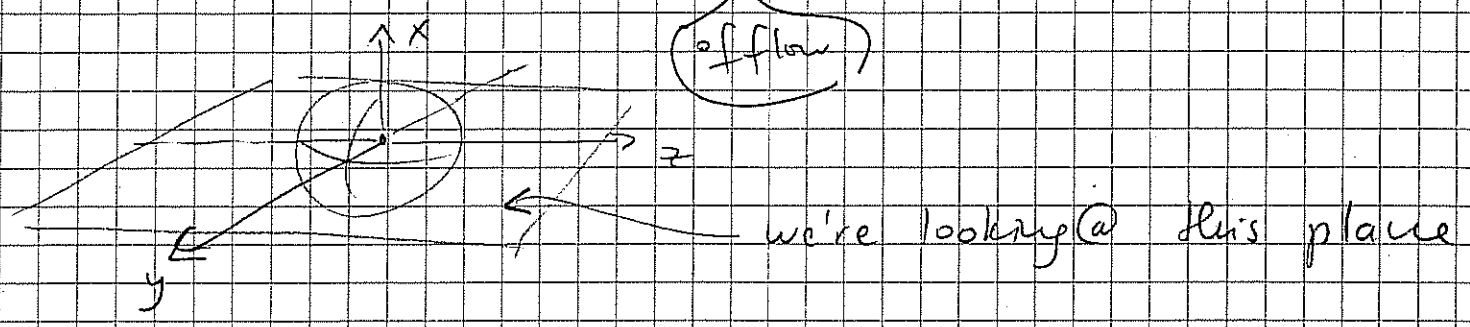
our Stokes' flow is stationary, so streamlines do not "wobble".

$$\frac{d\vec{r}(\tau)}{d\tau} = \vec{u}(\vec{r}(\tau))$$

We have axisymmetry.



so we'll take a slice @ $x=0$:



Furthermore, we have $y \leftrightarrow -y$ symmetry, so

Ok to look @ $y > 0$ only

velocity vector has u_z & u_y only & streamlines are found by solving:

$$\frac{dy(\tau)}{d\tau} = u_y(y(\tau), z(\tau))$$

$$\frac{dz(\tau)}{d\tau} = u_z(y(\tau), z(\tau))$$

Now recall in 2d incompressible flow we had introduced stream function:

$$\partial_x u_x + \partial_y u_y = 0 \Rightarrow u_x = \partial_y \psi(x, y), u_y = -\partial_x \psi(x, y)$$

so eqn. for streamlines $(x(\tau), y(\tau))$:

$$\frac{dx(\tau)}{d\tau} = u_x(x(\tau), y(\tau)) = \frac{\partial}{\partial y} \psi(x(\tau), y(\tau))$$

$$\frac{dy(\tau)}{d\tau} = u_y(x(\tau), y(\tau)) = -\frac{\partial}{\partial x} \psi(x(\tau), y(\tau))$$

$$\frac{dx}{d\tau} = \frac{\partial \psi}{\partial y}, \quad \frac{dy}{d\tau} = -\frac{\partial \psi}{\partial x} \Rightarrow d\tau = \frac{dx}{\partial \psi / \partial y} = -\frac{dy}{\partial \psi / \partial x}$$

⇒ streamline equ:

$$\frac{\frac{\partial \psi}{\partial y}}{\frac{\partial \psi}{\partial x}} = - \frac{dy}{dx}$$

or

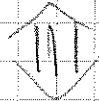
$$\frac{\partial \psi}{\partial x} dx = - \frac{\partial \psi}{\partial y} dy$$

or

$$\frac{\partial \psi}{\partial x} dx + \frac{\partial \psi}{\partial y} dy = 0$$

$$\text{or } d(\psi(x, y)) = 0$$

⇒ streamline in x, y -plane



lines where $\psi(x, y) = \text{const}$.
(since $d\psi = 0$ on streamlines)

MORAL:

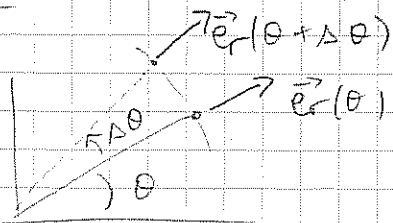
If $\psi(x, y)$ - stream function

then: curves of constant stream f'n = streamlines

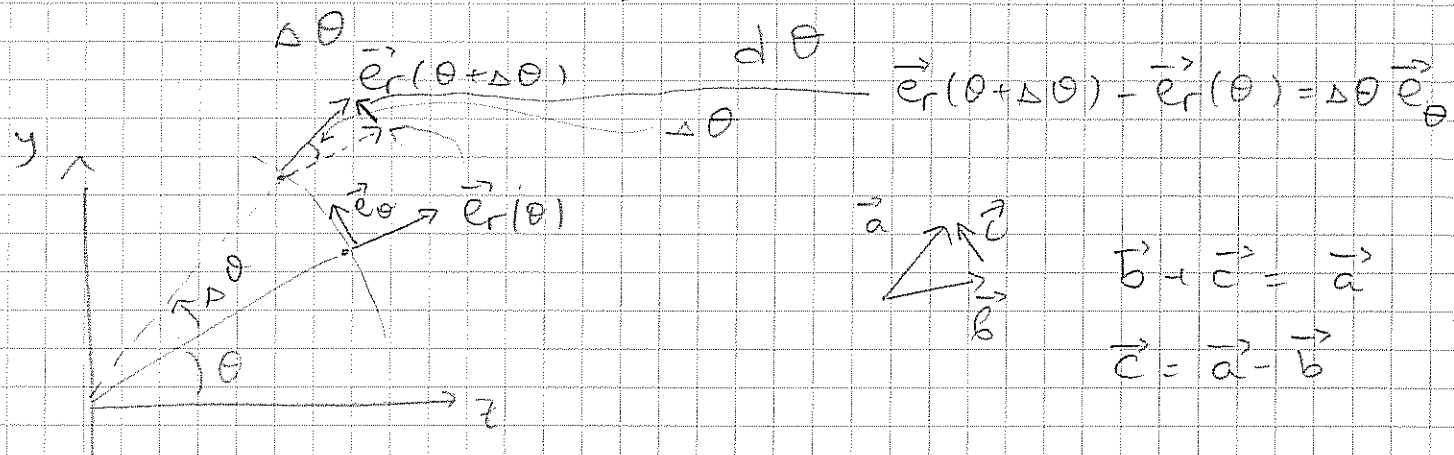
What about our (r, θ) -streamlines?

a slight subtlety: $(r(\tau), \theta(\tau)) \equiv$ streamline, clearly

Equ: $\vec{r}(\tau) = r(\tau) \vec{e}_r(\theta(\tau))$: 'cause \vec{e}_r depends on θ



$$\frac{\vec{e}_r(\theta + \Delta\theta) - \vec{e}_r(\theta)}{\Delta\theta} \equiv \frac{d\vec{e}_r}{d\theta}$$



$$\vec{b} + \vec{c} = \vec{a}$$

$$\vec{c} = \vec{a} - \vec{b}$$

$$\therefore \frac{d\vec{e}_r}{d\theta} = \vec{e}_\theta$$

so, in (r, θ) we have

$$\frac{d\vec{r}(t)}{dt} = \frac{dr}{dt} \vec{e}_r + r \frac{d\vec{e}_r}{d\theta} \frac{d\theta}{dt}$$

$$\frac{d\vec{r}}{dt} = \frac{dr}{dt} \vec{e}_r + r \vec{e}_\theta \frac{d\theta}{dt} \left(\begin{array}{l} \text{well} \\ \text{known: velocity} = \\ = \text{radial} + \text{angular} \\ (r\dot{r}) \quad (r\dot{\theta}) \end{array} \right)$$

$$\frac{d\vec{r}}{dt} = \vec{u} = u_r \vec{e}_r + u_\theta \vec{e}_\theta$$

hence: $\frac{dr}{dt} = u_r(r, \theta) \quad \rightarrow \quad \frac{dr}{u_r} = r \frac{d\theta}{u_\theta}$

$$r \frac{d\theta}{dt} = u_\theta(r, \theta)$$

Now $u_r = \frac{1}{r^2 \sin\theta} \frac{\partial \psi}{\partial \theta}$, $u_\theta = - \frac{1}{r \sin\theta} \frac{\partial \psi}{\partial r}$

hence streamline eqn is $\left(\frac{dr}{u_r} = r \frac{d\theta}{u_\theta} \right)$. (140)

$$\frac{dr}{u_r} = \frac{dr}{\frac{1}{r^2 \sin \theta} \frac{\partial \psi}{\partial \theta}} = r^2 \sin \theta \frac{dr}{\left(\frac{\partial \psi}{\partial \theta} \right)}$$

$$r \frac{d\theta}{u_\theta} = r \frac{d\theta}{-\frac{1}{r \sin \theta} \frac{\partial \psi}{\partial r}} = -r^2 \sin \theta \frac{d\theta}{\left(\frac{\partial \psi}{\partial r} \right)}$$

$$\text{or } r^2 \sin \theta \frac{dr}{\frac{\partial \psi}{\partial \theta}} = -r^2 \sin \theta \frac{d\theta}{\frac{\partial \psi}{\partial r}}$$

$$\Rightarrow \frac{\partial \psi}{\partial r} dr = - \frac{\partial \psi}{\partial \theta} d\theta \Rightarrow$$

$$\Rightarrow \frac{\partial \psi}{\partial r} dr + \frac{\partial \psi}{\partial \theta} d\theta = 0$$

$$\text{or } \underline{d(\psi(r, \theta)) = 0}$$

\Rightarrow streamlines \equiv lines of $\psi(r, \theta) = \text{const}$.

$$\psi(r, \theta) = \frac{U r^2 \sin^2 \theta}{4} \left(3 \frac{a}{r} - \left(\frac{a}{r} \right)^3 \right)$$

\uparrow
p131

$$\psi = \text{const} \equiv \frac{U}{4} r^2 \sin^2 \theta \frac{a}{r} \left(3 - \left(\frac{a}{r} \right)^2 \right)$$

$$\text{or } \left(r \left(3 - \left(\frac{a}{r} \right)^2 \right) \sin^2 \theta = \text{const} \right)$$

$$\psi = \frac{U}{4} r \left(3 - \frac{1}{r^2} \right) \sin^2 \theta = \text{const}$$

(a=1)

positive (0 - ∞)

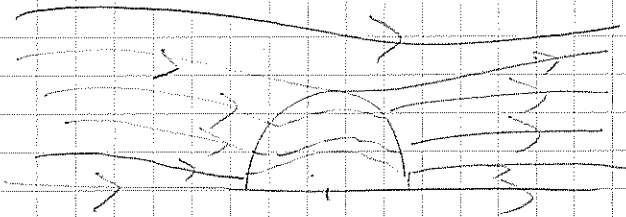
$$\left(3r - \frac{1}{r} \right) \sin^2 \theta = C$$

$$3r^2 \sin^2 \theta - \sin^2 \theta = Cr$$

$$3 \sin^2 \theta r - Cr - \sin^2 \theta = 0$$

$$r = \frac{1}{6 \sin^2 \theta} \left(C + \sqrt{C^2 + 12 \sin^4 \theta} \right)$$

This gives plot in Batchelor copied on p.132 ∴



⇒ misleading - frame where sphere is stationary
@ rest ---

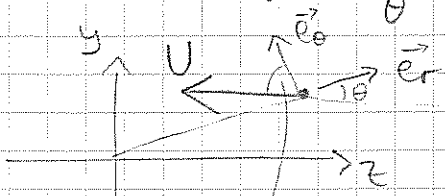
⇒ go to frame where sphere is @ rest

$$\vec{u}(r, \theta) \rightarrow -U \vec{e}_z + \vec{u}(r, \theta)$$

$\delta\psi = ?$

we know ψ for this

$$\left. \begin{aligned} -U \vec{e}_z &= -U \cos \theta \vec{e}_r \\ &+ U \sin \theta \vec{e}_\theta \end{aligned} \right\} \Rightarrow \int \begin{aligned} -U \cos \theta &= \frac{1}{r^2 \sin \theta} \partial_\theta \delta\psi \\ U \sin \theta &= -\frac{1}{r \sin \theta} \partial_r \delta\psi \end{aligned}$$



$\delta\psi = \text{streamf-n for } -U \vec{e}_z \text{ flow.}$

$$\partial_{\theta} \delta\psi = -U r^2 \sin\theta \cos\theta$$

$$\partial_r \delta\psi = -U r \sin^2\theta$$

$$\Rightarrow \delta\psi = -\frac{U r^2}{2} \sin^2\theta$$

check $\partial_{\theta} \delta\psi = -\frac{U r^2}{2} \partial_{\theta} \sin^2\theta = -\frac{U r^2}{2} 2 \sin\theta \cos\theta$

$$\partial_r \delta\psi = -U r \sin^2\theta$$

so we have

$$\psi + \delta\psi = \frac{U}{4} r \left(3 - \frac{1}{r^2}\right) \sin^2\theta - \frac{U r^2}{2} \sin^2\theta$$

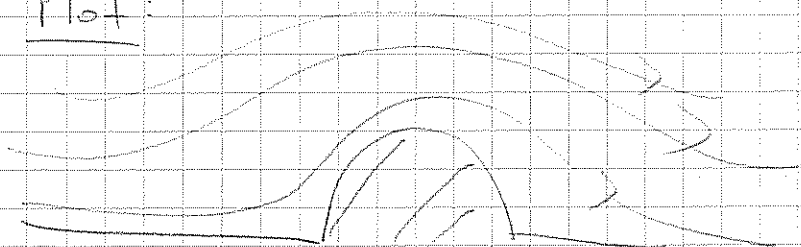
$$= 0 @ r=1, < 0 @ r > 1$$

$$= \frac{U}{4} \left[\left(-2r^2 + 3r - \frac{1}{r} \right) \sin^2\theta \right]$$

streamlines = contours of ψ .

so $\left(2r^2 - 3r + \frac{1}{r} \right) \sin^2\theta = \text{const.}$

Plot:



* see Mathematica plot, sphere of unit radius - streamlines for two cases.

MORAL:

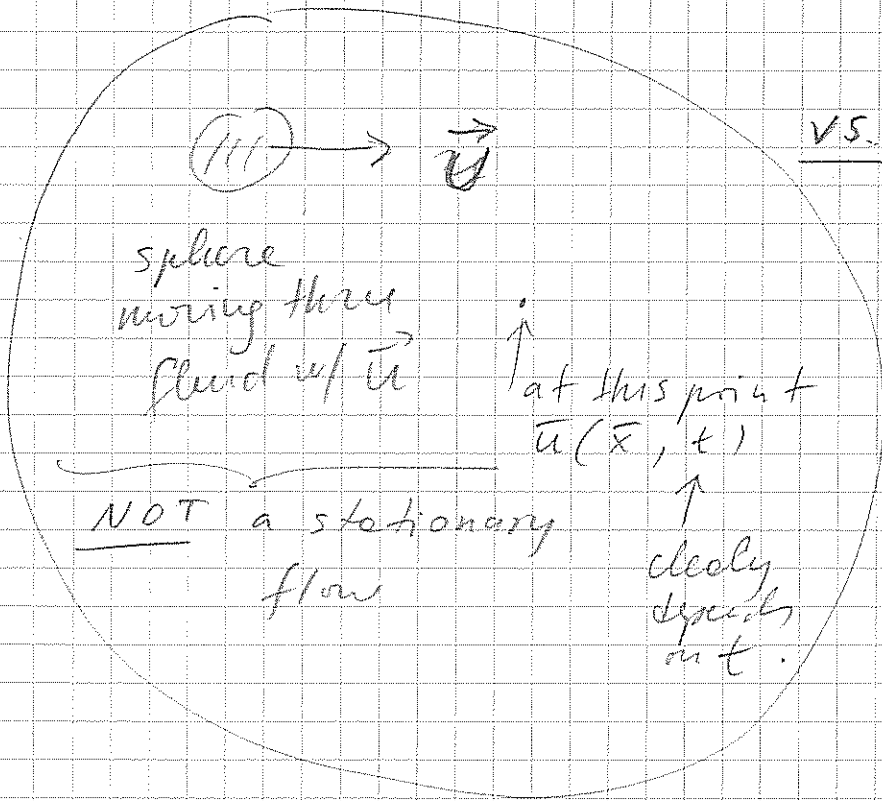
- care is always needed
- only for stationary flows

velocity field
pressure field — indep. of t

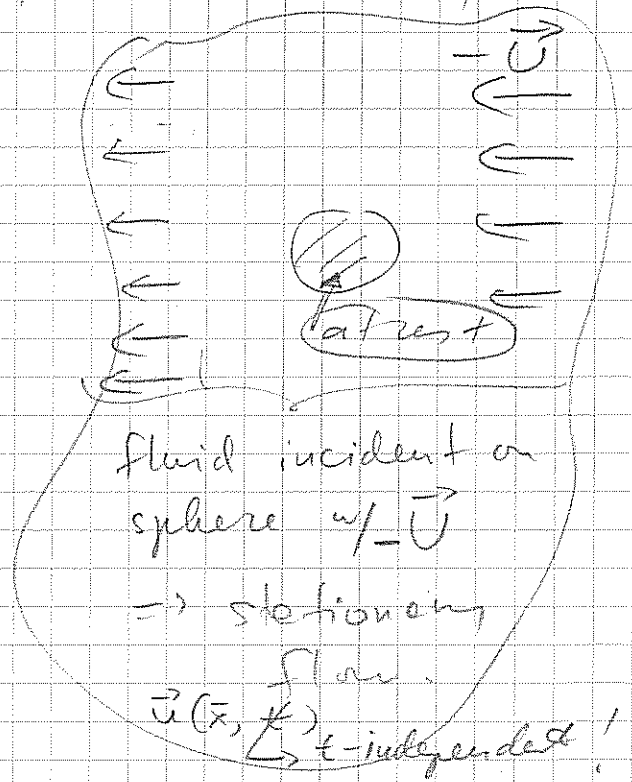
$$\vec{u} = \vec{u}(\vec{x})$$

$$p = p(\vec{x})$$

are streamlines = lines which physical fluid elements follow

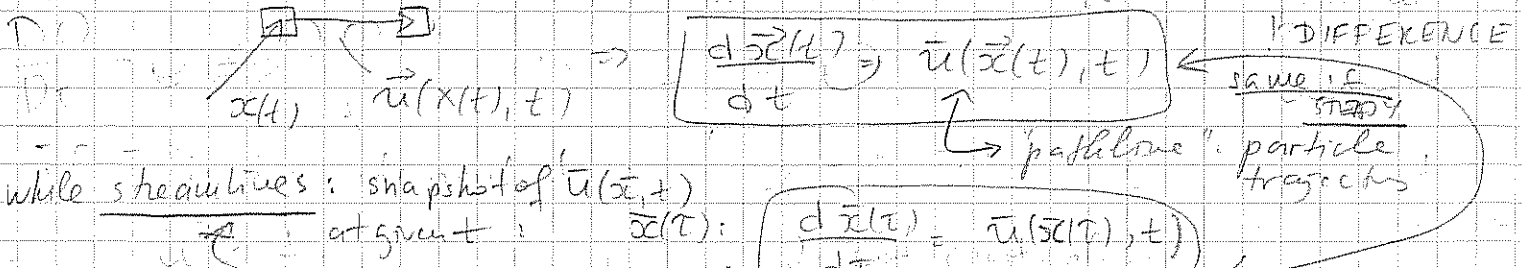


vs.



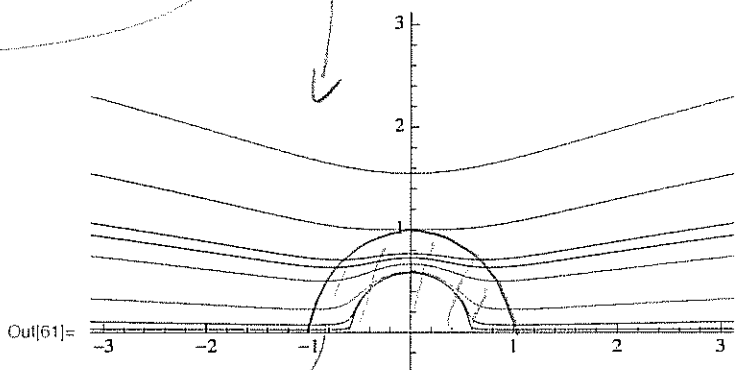
recall for physical fluid elements

$$\vec{x}(t+\Delta t) = \vec{x}(t) + \vec{u}(\vec{x}(t), t) \Delta t$$



streamlines
 $(3r - \frac{1}{r}) \sin^2 \theta = \text{const.}$

unsteady flow
 (moves w/ body)
 streamlines \neq pathlines

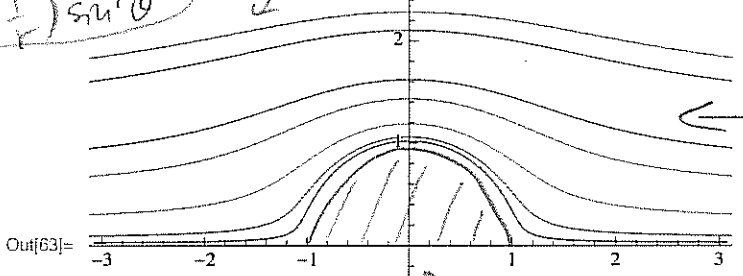


unphysical
 inside body -
 - but obey
 b.c. @ surface

\vec{u} : body moves w/ \vec{u}

streamlines
 $\text{const} = (2r^2 - 3r + \frac{1}{r}) \sin^2 \theta$

steady flow:
 streamlines =
 = pathlines



\vec{u}_{fluid}

at rest